

Saffman-Taylor instability in yield stress fluids: theory-experiment comparison

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Abstract: The Saffman-Taylor instability for yield stress fluids appears in various situations where two solid surfaces initially separated by such a material (paint, puree, concrete, yoghurt, glue, etc) are moved away from each other. The theoretical treatment of this instability predicts fingering with a finite wavelength at vanishing velocity, and deposited materials behind the front advance, but the validity of this theory has been only partially tested so far. Here, after reviewing the basic results in that field, we propose a new series of experiments in traction, to test the ability of this basic theory to predict data. We carried out tests with different initial volumes, distances and yield stresses of materials. It appears that the validity of the proposed instability criterion cannot really be tested under such experimental conditions, but at least we show that it effectively predicts the instability when it is observed. Besides, in agreement with the theoretical prediction for the finger size, a master curve is obtained when plotting the finger number as a function of the yield stress times the sample volume divided by the square initial thickness, in wide ranges of these parameters. This in particular shows that this traction test could be used for the estimation of the material yield stress.

I. Introduction

The Saffman-Taylor instability (STI) is observed when a fluid pushes a more viscous fluid in a confined geometry. The term confined here means that the distance between the solid walls is much smaller than the characteristic length in the flow direction. Such boundary conditions are typically encountered in porous media or between two parallel plates (i.e. Hele-Shaw cell). Under so-called “stable conditions” the length of the interface between the two fluids remains minimal, so that it is straight for a flow in a single direction, or circular for a radial flow. When the STI develops, the interface evolves in the form of fingers. For viscous fluids the origin of the instability is as follows: if the pressure along the interface is uniform, any perturbation or unevenness (local curvature) of the interface tends to develop further; this is so because the viscous fluid tends to advance faster in front of a curvature in the flow direction as the fluid volume to be pushed is smaller. The development of this perturbation may only be damped if surface tension, which, on the contrary, works against the deformation of the initial interface, is sufficient to counterbalance the above viscous effect. This instability has been widely studied for simple fluids [1-3].

Experiments with radial Hele-Shaw cells using non-Newtonian fluids have shown striking qualitative differences in the fingering pattern (see e.g. [4-5]). It was discovered that, when the high-viscosity fluid is viscoelastic, the interface grows along a narrow and very tortuous finger leading to branched, fractal patterns [6]. It was also shown that this viscous fingering pattern can be replaced by a viscoelastic fracture pattern for appropriate Deborah numbers [7-8]. On the theoretical side, the treatment of the Saffman-Taylor instability problem was revisited for viscoelastic or shear-thinning fluids. Wilson [9] considered an Oldroyd-B fluid which exhibits elasticity and the case of power-law fluids was treated by Wilson [9] for unidirectional flows and by Sader et al. [10] and Kondic et al. [11] for radial flows. However, except in the case of fluids with a negative viscosity for which slip layers may form [11] or for strongly viscoelastic fluids [8], the corresponding theoretical results did not show strong changes in the basic process of instability as it appears for Newtonian fluids. For viscoelastic fluids, Wilson [9] found a kind of resonance which can produce sharply increasing (in fact unbounded) growth rates as the relaxation time of the fluid increases. Sader et al. [10] mainly showed that decreasing the power-

51 law index dramatically increases the growth rates of perturbation at the interface and provides
52 effective length compression for the formation of viscous-fingering patterns, thus enabling them to
53 develop much more rapidly. For non-elastic weak shear-thinning fluids Lindner et al. [12] showed that
54 during the evolution of the Saffman-Taylor instability in a rectangular Hele-Shaw cell, the width of the
55 fingers as a function of the capillary number collapse onto the universal curve for Newtonian fluids,
56 provided the shear-thinning viscosity is used to calculate the capillary number. For stronger shear-
57 thinning, narrower fingers are found. Further observations on shear-thinning elastic materials were
58 provided by Lindner et al. [13].

59 As far as we know the theoretical description of STI with yield stress fluids (YSF), which can flow only
60 beyond a critical stress otherwise they behave as solids [14], starts with the work of Coussot [15]. For
61 both longitudinal and radial flows in Hele-Shaw cells. This approach is based on the use of an
62 approximate Darcy's law for yield-stress fluids, which leads to a dispersion equation for both flow types
63 similar to equations obtained for ordinary viscous fluids except that now the viscous terms in the
64 dimensionless numbers conditioning the instability contain the yield stress. As a consequence the
65 wavelength of maximum growth can be extremely small even at vanishing velocities, so that the STI
66 can still exist and we have an original situation: a "hydrodynamic" instability at vanishing velocity.
67 Another original aspect of this instability for YSF is that at sufficiently low flow rate the fingering
68 process leaves arrested fluid volumes behind the advancing front [15]. Miranda [16] presented a
69 theoretical analysis which goes beyond the above theory by using a mode-coupling approach to
70 examine the morphological features of the fluid-fluid interface at the onset of nonlinearity, and finally
71 proposed mechanisms for explaining the rising of tip-splitting and side-branching events. However this
72 approach relies on a Darcy-law-like equation valid in the regime of high viscosity compared to yield
73 stress effects. On the contrary, as we are interested in the specific effect of yielding we focus on
74 situations for which there is a major impact of the yield stress. On another side, numerical approach
75 was also developed to study the standard problem of penetration of a finger in a Hele-Shaw cell (for
76 Newtonian fluids a stationary finger forms), first for a simple YSF [17], then for a thixotropic fluid [18].

77
78 Experimentally the SFI of YSF has been studied in a rectangular Hele-Shaw cell with Carbopol gels [19-
79 20]. This relies on the injection of air at a given point in the middle of the cell, which then propagates
80 through the fluid. For a Newtonian viscous fluid, when the instability criterion is fulfilled, some fingers
81 develop in the cell but, after some distance, one finger becomes dominant while the others stop and
82 this single finger advances steadily along the main cell direction, with a size equal to half the cell width.
83 The result with a YSF is strongly different: at some time there can be one finger, but with a size possibly
84 much smaller than the cell width, but this finger will soon destabilize in secondary fingers which are
85 finally stop, leaving again one finger and so on. A comparison with theory is hardly possible in this
86 context, but the details of the evolution and the different regimes have been described [20]. Similar
87 approaches were also developed for thixotropic YSF [21], which obviously gives rise to effects more
88 complex to predict due to the time-dependency of the fluid behavior.

89
90 There is a situation in which the STI of YSF is currently observed: the separation of two plates initially
91 in contact with a thin layer of YSF; as the plates are moved away the layer thickness increases, which
92 induces a radial flow towards some central position; if the distance between the plates is sufficiently
93 small, the radial velocity is much larger than the axial one, so that the flow approximately corresponds
94 to a radial flow driven by the air entering the gap, which corresponds to the conditions under which
95 the STI can be considered. This is the most frequent situation under which the STI for YSF can be
96 observed in our everyday life: as soon as some thin layer of paint, glue, puree, yoghurt, is squeezed
97 between two solid surfaces (a tool, a spoon, etc) which are then separated, one observes a
98 characteristic fingering shape. Note that it is possible to observe such pictures because the fluid leaves
99 arrested regions behind the flow front, which finally give this definitive shape. This contrasts with

100 simple liquids for which the fingers soon relax under the action of wetting effects and a uniform layer
101 rapidly reforms.

102

103 Finally most of the theory-experiments comparisons concern the observations from traction tests. In
104 that case a reasonable agreement between the fingering wavelength and the theoretical predictions
105 was found [19, 22], but this was done in relatively narrow range of parameters, as essentially the gap
106 was varied. Besides, somewhat problematic are the observations of Barral et al. [23] which showed
107 that there is a strong discrepancy between the theoretical conditions and the experimental data
108 concerning the onset of the instability. The problem is that this appears to be the only experimental
109 approach of the onset of this instability with YSF, and it is in complete disagreement with existing
110 theory, which might suggest that something is missing in the theory.

111

112 Our present objective is to attempt to clarify the situation through new experiments and further
113 discussion of the experimental criterion of instability and the fingering wavelength. We rely on new
114 systematic traction tests under different conditions (fluid volume, initial aspect ratio, interaction with
115 the solid surface) and an analysis of these data with a critical eye, allowing to reach some clearer
116 conclusions about the validity of the theory.

117

118

119 **I. Materials and methods**

120 *Materials*

121 We used oil-in-water (direct) emulsions made of silicone oil (viscosity 0.35 Pa.s) as dispersed phase,
122 and a continuous phase (viscosity 5 mPa.s) made of distilled water and 3 wt%
123 myristyltrimethylammonium bromide (TTAB, Sigma-Aldrich). A Silverson mixer (model L4RT),
124 equipped with a rotating steel blade inside a punched steel cylinder, was also used as emulsifier. During
125 the preparation, the fluids are sheared and the oil phase is broken into small droplets while the water
126 or water/glycerol phase fills the surrounding environment, and the interface is stabilized by surfactants
127 (TTAB). The rotation velocity of the mixer is progressively increased to reach the maximum rate of 6000
128 rpm. A part of the bubbles incorporated in the mixture during this process can be removed by tapping
129 the container, the rest of the bubbles is removed by centrifugation. The droplet size is approximately
130 uniform, around 5 microns. Different emulsions with different oil volume fractions were prepared. The
131 resulting yield stress of the emulsions prepared at 76%, 78%, 82% and 84% (volume concentration of
132 oil), was respectively 20 Pa, 30 Pa, 40 Pa and 50 Pa, within 1 Pa.

133 We also used a Carbopol (U980) gel. It has been observed that this material is essentially a glass made
134 of a high concentration of individual, elastic sponges (with a typical element size of 2 to 20 μm) [24],
135 which gives rise to its yielding behavior. The preparation of Carbopol gel begins with the introduction
136 of some water in a mortar mixer. The rotation velocity is set at 90 rpm and the appropriate amount of
137 raw Carbopol powder (1%wt) is slowly added to the stirring water. After about one hour, the
138 incorporation of the powder is done and the appropriate amount of Sodium Hydroxide (1 mol/L) is
139 quickly added to the solution, which increases its pH. The mixing is then maintained for approximately
140 one day to allow a full homogenization of the mixture.

141

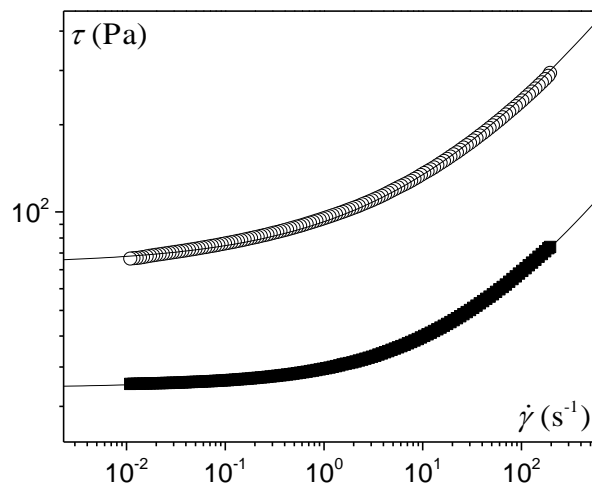
142 *Rheological characterization*

143 Rheological tests were performed with a Kinexus Malvern-stress-controlled rheometer equipped with
144 two circular, rough plates (diameter: 40 mm). The sample was carefully set up and the gap was fixed
145 at 2 mm taking care not to entrap air bubbles. A logarithmically increasing then decreasing stress ramp

146 test was then applied over a total time of four minutes. Except for the first part of the increasing curve
 147 associated with deformations in the solid regime the increasing and decreasing shear stress vs shear
 148 rate curves superimpose. We retain, here, the decreasing part as the flow curve of the material. For
 149 similar emulsions it has been shown that this apparent flow curve obtained from macroscopic
 150 observations correspond to the effective, local constitutive equation observed at a local scale with
 151 imaging technique [25]. The emulsions and the Carbopol gel exhibit a simple yield stress fluid
 152 behaviour and their flow curve can be well fitted by a Herschel-Bulkley (HB) model (see typical results
 153 in Figure 1):

$$154 \quad \tau > \tau_c \Rightarrow \tau = \tau_c + k\dot{\gamma}^n \quad (1)$$

155 in which τ is the shear stress, $\dot{\gamma} > 0$ the shear rate, τ_c the yield stress, k the consistency factor and
 156 n the power-law exponent.



157
 158 **Figure 1:** Typical flow curve of a one of our emulsion and of the Carbopol gel. The
 159 continuous lines are the Herschel-Bulkley model fitted to data with the parameters:
 160 (emulsion (78%)) $\tau_c = 30 \text{ Pa}$, $k = 4.5 \text{ Pa}\cdot\text{s}^n$ and $n = 0.45$; (Carbopol) $\tau_c = 70 \text{ Pa}$,
 161 $k = 23.5 \text{ Pa}\cdot\text{s}^n$ and $n = 0.4$.

162
 163 **Set up for traction tests**

164 For the adhesion tests a dual-column testing system (*Instron* model 3365) with a position resolution of
 165 $0.118 \mu\text{m}$ was used. The column was equipped with either a 10 or 500 N static load cell which were
 166 able to measure the force to within a relative value of $\pm 10^{-6}$ of the maximum value. Waterproof
 167 sandpaper (average particle diameter $82 \mu\text{m}$, a dimension much larger than the typical droplet size)
 168 was attached to the top and bottom plates. Since the volume loss in the roughness could be significant
 169 in some cases, a generous amount of extra sample was applied to the surface of the sandpaper before
 170 each test and the excess removed by scraping the surface with a palette knife. This also ensured
 171 reproducible wetting conditions of the fluid onto the solid surface. However qualitatively similar
 172 results were obtained with initially dry or wet surfaces. Between two successive tests, both plates were
 173 removed and cleaned. A given volume (Ω_0) of material was then collected with a syringe, put at the
 174 center of the bottom plate, and the upper plate was decreased at a fixed (initial) height (h_0), thus
 175 squeezing the material. The adhesion test then consisted of lifting the upper plate at a constant
 176 velocity (0.01 mm/s) while monitoring the force (F) applied to the upper plate. The initial distance
 177 was varied between 0.2 mm and 5 mm . The initial volume was varied between 0.3 and 3 ml .

178

179 II Theoretical

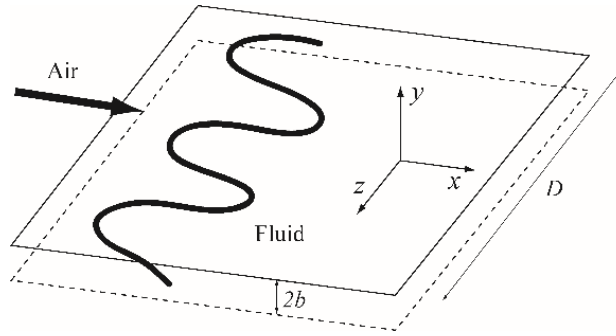
180 II.1 Instability in a straight Hele-Shaw flow

181

182 The instability of radial flows of Newtonian fluids in Hele-Shaw cells has been studied [3, 26-27] by
183 using the vectorial form of Darcy's law. The treatment below summarizes the assumptions and results
184 of Coussot [15], whose approach has some similarity with the one adopted by Wilson [9] or Sader et
185 al. [10] who considered power-law fluids and could not directly use the standard (Newtonian) Darcy's
186 law.

187

188 We consider a yield stress fluid pushed by an inviscid fluid (say, air) so that it tends to flow in a given
189 direction x between two parallel plates separated by a distance $h = 2b$, with a mean fluid velocity U
190 . The initial interface is assumed to be uniform and straight (along the z direction). A stable flow
191 corresponds to a fluid motion along the x direction, uniform along the z direction. For an unstable
192 flow this interface does not remain straight (see Figure 2).
193



194

195

196 **Figure 2:** Scheme of main flow characteristics in a Hele-Shaw cell for a destabilized
197 interface (thick line).
198

199 The linear stability analysis of this flow [15] relies on several assumptions: (i) the constitutive equation
200 of the fluid can be well represented by a HB behaviour; (ii) the lubrication assumption is valid, i.e. the
201 velocity component perpendicular to the cell plan can be neglected; (iii) the shear stress at the wall,
202 even around the front of the flow, can be approximated by a value close to the exact one for a stable
203 uniform flow through this cell (see [15]):

$$204 \quad \tau_w = \tau_c \left[1 + c \left(\frac{kU^n}{\tau_c b^n} \right)^d \right] \quad (2)$$

205 where c and d two parameters which depend on n . For example, for $n=1/3$, $c=1.93$, $d=0.9$ [15].
206

207 Under these conditions the linear stability analysis of the flow, for negligible gravity effects, predicts
208 that the unidirectional flow above described is fundamentally unstable as soon as the inviscid fluid
209 pushes the yield stress fluid. Moreover, the wavelength of maximum growth is

$$210 \quad \lambda_m = 2\pi \sqrt{\frac{3\sigma b}{\tau_w}} \quad (3)$$

211 in which σ is the surface tension. Note that the Newtonian case is recovered from this approach: by
212 using in equation (3) the wall stress expression for a stable uniform flow of a Newtonian fluid in such
213 a cell, i.e. $\tau_w = 3\mu U/b$, we find $\lambda_m = 2\pi b \sqrt{\sigma/\mu U}$, which is the standard expression found from a
214 complete theoretical analysis in the Newtonian case [2].
215

216 From (3) we also deduce that the instability will be apparent only if $\lambda_m < D$, where D is the width of
 217 the flow. This implies that the flow will be apparently unstable if

$$218 \quad \tau_w > \frac{12\pi^2 \sigma b}{D^2} \quad (4)$$

219
 220 At last note that for a yield stress fluid, τ_w tends to τ_c when $U \rightarrow 0$ or, more precisely, $\tau_w \approx \tau_c$ when
 221 $kU^n / \tau_c b^n \ll 1$. Thus, at vanishing velocity, the wavelength tends to a finite value, i.e. $\lambda_m = 2\pi \sqrt{3\sigma b / \tau_c}$
 222 . This strongly contrasts with the result of the Saffman-Taylor instability for simple fluids (i.e. without
 223 yield stress) for which the wavelength tends to infinity when the velocity tends to zero. Thus, for yield
 224 stress fluid, if the front width is sufficiently large, we will see the development of a hydrodynamic
 225 instability at vanishing velocity. Note that, more precisely, due to the square root of the stress in the
 226 wavelength expression, the approximation above leading to neglect the flow rate dependent term in
 227 the stress expression, leads to an approximation to within 10% of the exact value of the wavelength if
 228 $kU^n / \tau_c b^n$ is smaller than 0.2.

229
 230 Moreover, in the case of small front velocity, the stress should slightly overcome the yield stress in the
 231 regions with highest velocities and, as a consequence, intuitively, the stress might be smaller than the
 232 yield stress in regions with lowest velocities (see further demonstration in [15]). As a consequence the
 233 regions left behind should remain static just after the beginning of the unstable process. As long as the
 234 fingers grow, the pressure drop applied to these regions therefore decreases so that they should
 235 remain static even after a long time.

236
 237

238 II.2. Instability in a radial Hele-Shaw flow

239

240 We consider now the case of a radial flow, with an inviscid fluid pushing the yield stress fluid towards
 241 the center. This assumes that, if the plates remain at the same distance, the YSF for example escapes
 242 through a central hole. Using again the expression (2) for the wall shear stress (which neglects
 243 orthoradial components), a linear stability analysis [15] leads to

$$244 \quad \lambda_m = 2\pi R \left(\frac{3\sigma b}{\sigma b + \tau_w R^2} \right)^{1/2} \quad (5)$$

245 in which R is the radius of the circular interface. Once again this expression allows to recover the
 246 Newtonian case, $\lambda_m = 2\pi R / \sqrt{\mu UR^2 / \sigma b^2 + 1/3}$ [3], by introducing in (5) the expression for the wall
 247 shear stress of the stable, uniform flow of a Newtonian fluid (see above).

248

249 Finally, for a YSF, the criterion for the apparent onset of instability ($\lambda_m < 2\pi R$) is:

$$250 \quad \tau_w > \frac{2\sigma b}{R^2} \quad (6)$$

251 The above remarks concerning the finite wavelength at vanishing velocity and the tracks left behind
 252 still apply in this case.

253

254

255 II.3. Flow induced by a traction test

256

257 We now consider the flow induced by a traction test, in which the material initially forming a cylindrical
 258 layer situated between two plates, is then deformed as result of the relative motion of the two plates
 259 away from each other along their common axis. As the distance between the plates increases, since
 260 the material remains in contact with the plate, the thickness of the sample increases. As a result the
 261 material tends to gather towards its central axis. Let us consider the ideal case where the sample shape

262 remains cylindrical during this process, i.e. the flow is stable and we neglect the deposits of material
 263 along its motion along the plates. In that case, as a result of mass conservation, the mean radial velocity
 264 (U) is related to the velocity of separation of the plates (V) through

$$265 \quad U = \frac{R}{4b} V \quad (7)$$

266 From (7) we see that as soon as the aspect ratio (i.e. $R/2b$) of the sample is sufficiently large, the radial
 267 velocity is much larger than the separation velocity. In that case the lubrication assumption, i.e. the
 268 velocity components parallel to the plates are much larger than the perpendicular ones, is relevant,
 269 and we can consider that the flow is similar to that resulting from a pure radial flow between plates at
 270 a fixed distance from each other. Obviously this assumption will start to fail at some point during the
 271 process, as the aspect ratio progressively decreases toward smaller values when the plates are moved
 272 away from each other. In the following we will a priori assume that the lubrication assumption is valid,
 273 and discuss its possible non-validity as an artefact of the tests.

274
 275 On the other side, for such a traction test, we can easily estimate the normal force needed to separate
 276 the plates under the lubrication assumption for stable and sufficiently slow flows (i.e. $kU^n / \tau_c b^n \ll 1$)
 277 [28]. In that case the radial flow along the plate induces a shear stress equal to the material yield stress.
 278 The momentum balance applied to the sample volume between R and r assuming no surface tension
 279 effect and negligible atmospheric pressure leads to:

$$280 \quad p(r) = \frac{\tau_c}{b} (r - R) \quad (8)$$

281 The net normal force exerted onto the plate in that case is then found by integrating the pressure (8)
 282 over the surface of contact:

$$283 \quad F = \frac{2\pi\tau_c R^3}{3b} \quad (9)$$

284 Equation (9) thus provides an expression for the force applied in the case of slow flows. Since the
 285 assumed constitutive equation is continuous, i.e. it predicts a continuous transition from rest to slow
 286 flows around the yield stress, equation (9) also provides an expression for the minimum force to induce
 287 some motion for a given separation distance and a given radius.

288 Note that for a given volume of material ($\Omega_0 = 2\pi R^2 b$) this force may be rewritten as

$$289 \quad F = \frac{4\tau_c \Omega_0^{3/2}}{3\sqrt{\pi h^{5/2}}} \quad (10)$$

290 which gives the force variation as a function of the distance ($h = 2b$) between the plates.

291

292

293 **III Results and discussion**

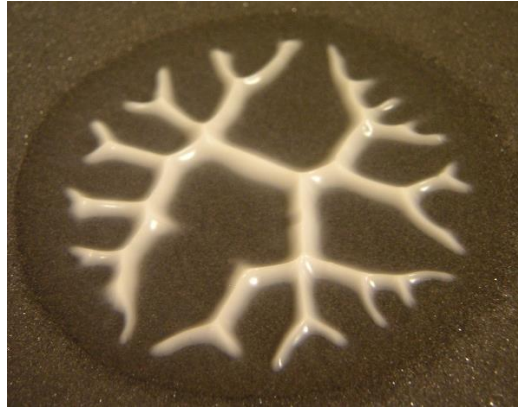
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295 **III.1 General trends**

296

297 The typical result of a traction test is the formation of an approximately symmetrical deposit over each
 298 solid surface, associated with a tendency to a gathering of the material towards the central axis, as
 299 proved by the larger thickness of material towards the central part. Depending on the experimental
 300 conditions the final deposit has different aspects, from a simple conical shape to a fingered structure
 301 (see [23]). Since the initial shape is cylindrical, a stable flow would maintain a cylindrical interface. As
 302 a consequence a final fingered structure (see Figure 3) is the hallmark of flow instability.

303



304

305 **Figure 3:** Fingering aspect of the remaining deposit of yield stress fluid after separation of
 306 two plates initially squeezing a thin fluid layer. [Courtesy Q. Barral]

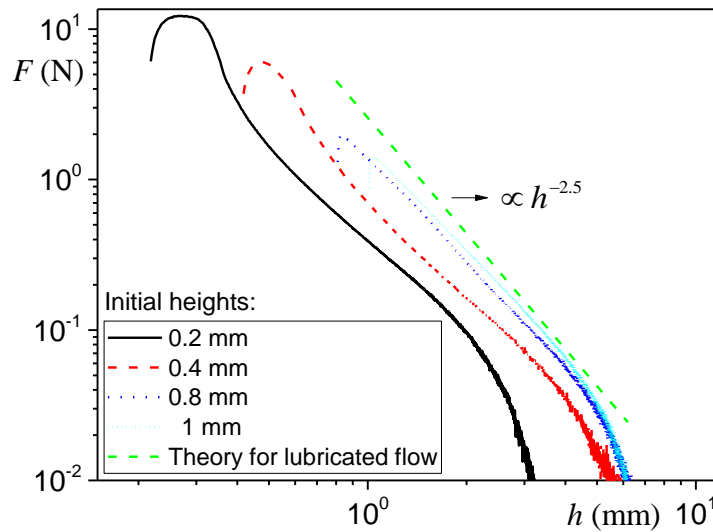
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308 III.2 Force vs distance

309

310 The force during such a test strongly decreases with the distance and approximately follows a slope of
 311 -2.5 in logarithmic scale (see Figure 4), which tends to confirm the validity of equation (10). However
 312 we can remark that the force curve is shifted towards smaller values when the initial distance is
 313 smaller, in contradiction with the above theory since the expression (10) only depends on the sample
 314 volume and the current distance. This is explained by the development of fingering, which implies that
 315 some significant parts of the material do not flow anymore in the radial direction.

316



317

318 **Figure 4 :** Force vs distance during a traction experiment for an emulsion (82%) (yield stress
 319 of 40 Pa) for different initial heights (and thus different aspect ratios) and same sample
 320 volume ($\Omega = 3 \text{ ml}$). The dashed line is the lubrication model (see text).

321

322

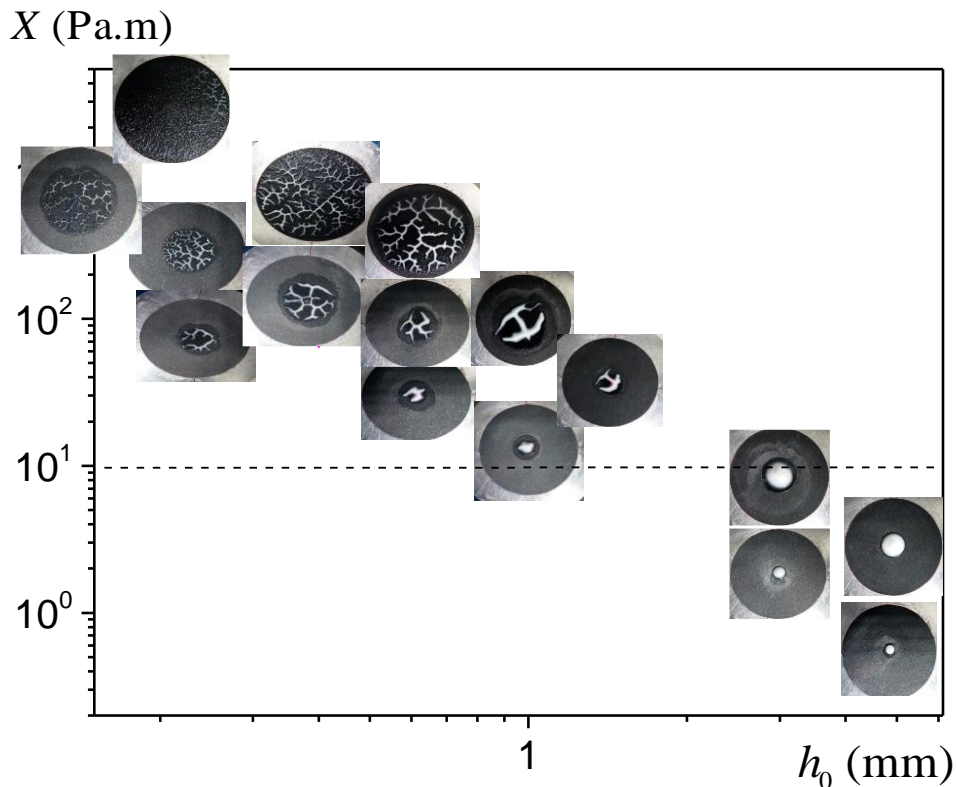
323 Let us try to take this phenomenon into account. We assume that during the withdrawal of dR , the
 324 fingers well develop so that half the material is left behind as deposited material, while the central
 325 flowing region is “plain”, with a current volume Ω . Thus we have a variation of the current volume of
 326 material still in the flowing region as $d\Omega = \pi R h dR$. Since by definition of this volume : $\Omega = \pi R^2 h$, we
 327 deduce $d\Omega/\Omega = dR/R$ and by integration $\Omega = \Omega_0 R/R_0 = \Omega_0^2 / \pi h R_0^2$. We finally find

328 $F_c = \frac{2\tau_c \Omega_0^{3/2}}{3\pi^{1/2}} h_0^{3/2} h^{-4}$. Although this expression now effectively predicts a decrease of the force with
 329 the initial distance, it also predicts a decrease of this force with the current distance as a power -4, in
 330 contradiction with the data. Thus we can conclude that although we are able to reproduce some
 331 qualitative trends through different approaches we still lack a full theory for describing the force
 332 evolution with distance when fingering develops significantly

335 III.3 Characteristics of the instability

337 *Instability criterion*

338
 339 We now discuss the characteristics of this instability. In order to better discuss the origin of the
 340 evolution of the final shape of the deposit with the material and process parameters we consider the
 341 theoretical prediction of the instability criterion (i.e. equation (6)) under negligible “additional viscous
 342 effects”. Note that we checked that for all the tests with the emulsions $kU^n/\tau_c b^n$ was smaller than
 343 0.2, which means that the above simplified expression for the finger width is relevant. This was not the
 344 case for the Carbopol gel, for which $kU^n/\tau_c b^n$ was as large as 0.5 at the beginning of the test in some
 345 cases, but in the following we neglect this aspect and it appears that this does not affect the
 346 consistency of our results and analysis. In the instability criterion (equation 6) we can then use the
 347 approximation $\tau_w \approx \tau_c$, so that this criterion may be rewritten as $\tau_c \Omega_0 / h^2 > 2\pi\sigma$. From this
 348 expression we see that, if it is to occur, the instability will occur at the beginning of the withdrawal,
 349 when the height is the smallest. As a consequence the instability criterion writes:
 350 $X = \tau_c \Omega_0 / h_0^2 > 2\pi\sigma$. Under these conditions we can expect that the instability will be “more
 351 developed” for increasing values of X .
 352



354 **Figure 5** : Final aspects of the deposit after plate separation as a function of the parameter
355 X and the initial thickness, for a 40 Pa yield stress emulsion and different sample volumes
356 (the volume corresponding to each picture may be estimated from the value of X and h_0
357 through $\Omega_0 = Xh_0^2/\tau_c$). The diameter of the dark disk in all the photos is 10 cm.

358
359
360 In figure 5 we show the different final shapes observed for different values of X as a function of the
361 initial distance between plates. We see that h_0 does not determine solely the intensity of the
362 instability: various deposit aspects are found for a given h_0 value. On the contrary, as expected from
363 the theory, the aspect of the deposits seems to be close for a given value of X : we get approximately
364 similar branched structures along each horizontal line in this representation (see Figure 5). Then we
365 can determine the limit between the unstable and stable regimes, by considering that the absence of
366 apparent fingering is the hallmark of stable flows. Note that for small values of h_0 this is an
367 extrapolation, since we were unable to get experimental data in this region of the graph as it required
368 too small sample volumes. We thus find that this limit corresponds to $X \approx 10$ Pa.m. [Note that a similar
369 approach from the data of Barral et al. [15] would lead to $X \approx 80$ Pa.m.] On the other side, using for
370 the surface tension the value of the interstitial liquid (water) [29], i.e. $\sigma = 0.07$ Pa.m, the left hand-
371 side of the instability criterion (6), i.e. $2\pi\sigma$, is equal to 0.4 Pa.m. Thus we find that experiments give
372 stable flows in a wide range where unstable flows are expected from theory, namely between say
373 $X \approx 10$ Pa.m and $X \approx 0.4$ Pa.m.

374
375 At first sight this result may be seen as a strong discrepancy between experiments and theory. Actually
376 this is not so obvious. Indeed, looking at the flows considered as “stable” we see that they correspond
377 to a rather limited radial flow. Under such conditions the STI, or more precisely fingering, would simply
378 not have enough time or distance to develop. More precisely, this is the validity of the lubrication
379 assumption which should be discussed. Indeed, we expect that if this assumption is valid, during the
380 traction test the distance will now increase to large values, i.e. at least of the same order as the initial
381 radius, which implies a significant radial flow. If we compute the ratio h_0/R_0 we find that the stable
382 flows correspond exactly to those for which h_0/R_0 is larger than 0.1. This suggests that here we in fact
383 find stable flows because the lubrication assumption is not valid. In that case we have indeed a more
384 complex flow than assumed in the theory, in particular there is now likely a significant component of
385 elongation along the vertical axis, which might damp the instability. In fact, looking further at the initial
386 and final sample shape we see that the material essentially transforms from a circular disk layer to a
387 cone of same basis, and does not significantly flow radially. Actually, in similar experiments carried out
388 with smooth plates, with roughness less than one nanometer, no instability at all is observed in a wide
389 range of initial distances [30]. In that case we have a pure elongation flow along most of the flow. This
390 further confirms the above suggestion that when the elongational component is significant no
391 instability can be expected.

392
393 This suggests that we cannot really test the validity of the instability criterion, because well before
394 the range for which stability is expected the lubrication assumption is not valid. We can just say that
395 under the proper assumptions, i.e. lubrication assumption, the flow is unstable, in agreement with the
396 theoretical criterion.

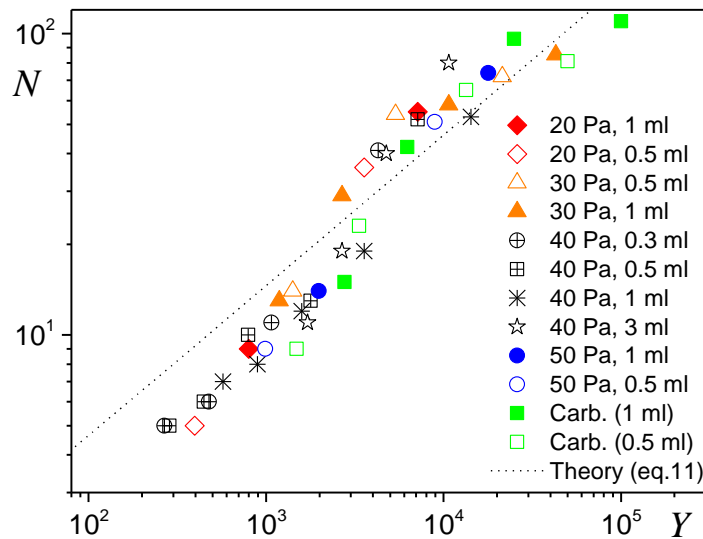
397 *Fingering wavelength*

398
399
400 Let us now try another approach to test the theory, by looking at the wavelength of the fingering. In
401 that aim we need to look at the fingering characteristics at the onset of the instability, since this is the
402 only aspect relevant within the frame of a linear stability analysis which considers the flow evolution
403 for a slight perturbation of the initial stable flow. We assume that the corresponding wavelength

404 corresponds to the fingers apparent at the periphery of the initial sample layer and we simply count
 405 the number of deposited fingers around the sample, which corresponds to $N/2$. The theoretical
 406 prediction for the finger number N at the onset of the instability, i.e. $2\pi R_0/\lambda_m$, as deduced from (5)
 407 writes:

$$408 \quad N = \frac{1}{\sqrt{3}} \left(1 + \frac{2Y}{\pi} \right)^{1/2} \quad (11)$$

409 in which $Y = X/\sigma$. All the data are shown in Figure 6, where we can see that globally the theoretical
 410 prediction is in agreement with the experiments: all the data for the different fluids, different initial
 411 distances, and different sample volumes, fall along a master curve which corresponds to an increase
 412 of the finger number following on average the theoretical curve. Nevertheless we can note some
 413 discrepancy: at low Y values the finger number is systematically below (by a factor about 1.5) the
 414 theoretical prediction, which suggests that in that case the elongational component of the flow plays
 415 some significant role and slightly damps the instability; at large Y the finger number is slightly above
 416 the theoretical prediction, but we have no explanation for that.
 417



418 **Figure 6** : Saffman-Taylor instability during traction tests with emulsions of various yield
 419 stresses, for various sample volumes and various initial thicknesses: Number of fingers as a
 420 function of the parameter Y . Caption: emulsions, except when mentioned (Carbopol).
 421

422
 423
 424

Conclusion

425 We have shown that it is in fact not possible to test the theoretical criterion for the instability onset of
 426 yield stress fluids in traction tests: as soon as the radial flow induced is sufficiently developed an
 427 unstable flow is expected, and effectively observed. On the contrary, in agreement with the theoretical
 428 prediction for the finger size appears, i.e. a master curve is obtained when plotting N as a function of
 429 Y for the different volumes, yield stresses and initial thicknesses in rather large ranges. However,
 430 there remains a discrepancy between theory and experiments: the finger number is smaller by a factor
 431 about 1.5 for $Y < 350$, and larger by a factor about 1.5 for $Y > 350$. As a consequence, the basic theory
 432 can be used to estimate and predict the fingering aspect for any application, as soon as one knows the
 433 material yield stress, sample volume and initial thickness. On the other side, this traction test could be

434 used for the estimation of the material yield stress, from an analysis of the fingering characteristics.
435 Further work in that field could focus on the exact flow characteristics during such a traction test, in
436 particular when the instability has begun.

437

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